

# Harmonic Excitation of Damped Systems

## Section 2.2

## Motorcycle Engine Vibration Problem

- A motorcycle engine turns (and vibrates) at **300 rpm** with a harmonic force of **20 N**.
- What is the amplitude of the vibration with respect to the frame (assumed to be stationary) and phase of the response (with respect to the force), if:
  - the mass of the engine is **40 kg**,
  - the stiffness of the mounts is **40 kN/m** and
  - the damping coefficient is **125 Ns/m**?

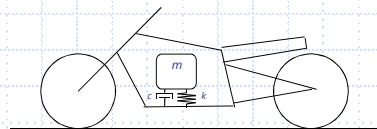
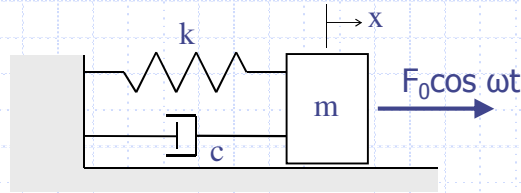


Figure 1. Motor vibrating in motorcycle

## Damped Spring-Mass System with Forced Vibration



$$m\ddot{x} + c\dot{x} + kx = F_0 \cos \omega t$$

Dividing both sides by  $m$  yields:

$$\ddot{x} + 2\zeta\omega_n\dot{x} + \omega_n^2 x = f_0 \cos \omega t$$

where:  $f_0 = \frac{F_0}{m}$

## Solving the Differential Equation

Assume a particular solution:

$$x_p(t) = A \cos \omega t + B \sin \omega t$$

so that:  $\dot{x}_p(t) = -\omega A \sin \omega t + \omega B \cos \omega t$

$$\ddot{x}_p(t) = -\omega^2 (A \cos \omega t + B \sin \omega t)$$

Substituting into the D.E.:

$$\begin{aligned} & -\omega^2 (A \cos \omega t + B \sin \omega t) + 2\zeta\omega_n\omega(-A \sin \omega t + B \cos \omega t) \\ & + \omega_n^2 (A \cos \omega t + B \sin \omega t) = f_0 \cos \omega t \end{aligned}$$

## Solving the Differential Equation

Separating  $\cos \omega t$  and  $\sin \omega t$  terms yields:

$$\begin{aligned} &(-\omega^2 A + 2\zeta\omega_n\omega B + \omega_n^2 A - f_0)\cos \omega t \\ &+ (-\omega^2 B - 2\zeta\omega_n\omega A + \omega_n^2 B)\sin \omega t = 0 \end{aligned}$$

In order to equal zero all the time, both of the constants in front of the  $\cos \omega t$  and  $\sin \omega t$  terms must equal zero. I.e.:

$$\begin{aligned} (\omega_n^2 - \omega^2)A + 2\zeta\omega_n\omega B &= f_0 \\ (-2\zeta\omega_n\omega)A + (\omega_n^2 - \omega^2)B &= 0 \end{aligned}$$

## Solving the Differential Equation

- Solving for  $A$  and  $B$  yields:

$$A = \frac{(\omega_n^2 - \omega^2)f_0}{(\omega_n^2 - \omega^2)^2 + (2\zeta\omega_n\omega)^2} \quad B = \frac{2\zeta\omega_n\omega f_0}{(\omega_n^2 - \omega^2)^2 + (2\zeta\omega_n\omega)^2}$$

- Substituting for  $A$  and  $B$  in  $x_p(t)$  yields:

$$x_p(t) = \frac{f_0}{(\omega_n^2 - \omega^2)^2 + (2\zeta\omega_n\omega)^2} ((\omega_n^2 - \omega^2)\cos \omega t + 2\zeta\omega_n\omega \sin \omega t)$$

## Solving Differential Equation

- Re-writing in terms of  $X$  and  $\theta$  instead of  $A$  and  $B$  yields:

$$x_p(t) = \frac{f_0}{\sqrt{(\omega_n^2 - \omega^2)^2 + (2\zeta\omega_n\omega)^2}} \cos\left(\omega t - \tan^{-1}\left(\frac{2\zeta\omega_n\omega}{\omega_n^2 - \omega^2}\right)\right)$$

- Final equation** (for **forced vibration** without free vibration):

$$x_p(t) = X \cos(\omega t - \theta)$$

where:  $X = \frac{f_0}{\sqrt{(\omega_n^2 - \omega^2)^2 + (2\zeta\omega_n\omega)^2}}$

$$\theta = \tan^{-1}\left(\frac{2\zeta\omega_n\omega}{\omega_n^2 - \omega^2}\right)$$

## Solving Differential Equation

- If we consider both the **free vibration** and **forced vibration**  $x(t) = x_h(t) + x_p(t)$ , solving for amplitude components in terms of  $x_0$  and  $v_0$  yields:

$$x(t) = e^{-\zeta\omega_n t} \left\{ \left( x_0 - \frac{f_0(\omega_n^2 - \omega^2)}{(\omega_n^2 - \omega^2)^2 + (2\zeta\omega_n\omega)^2} \right) \cos \omega_d t + \left( \frac{\zeta\omega_n}{\omega_d} \left( x_0 - \frac{f_0(\omega_n^2 - \omega^2)}{(\omega_n^2 - \omega^2)^2 + (2\zeta\omega_n\omega)^2} \right) - \frac{2\zeta\omega_n\omega^2 f_0}{\omega_d [(\omega_n^2 - \omega^2)^2 + (2\zeta\omega_n\omega)^2]} + \frac{v_0}{\omega_d} \right) \sin \omega_d t \right\}$$

$$+ \frac{f_0}{(\omega_n^2 - \omega^2)^2 + (2\zeta\omega_n\omega)^2} [(\omega_n^2 - \omega^2) \cos \omega t + 2\zeta\omega_n\omega \sin \omega t]$$

## Plotting the “Frequency Response”

- Looking at the forced vibration  $x_p(t)$ , we can plot the amplitude  $X$  and phase lag  $\theta$  as a function of forcing frequency  $\omega$ .

