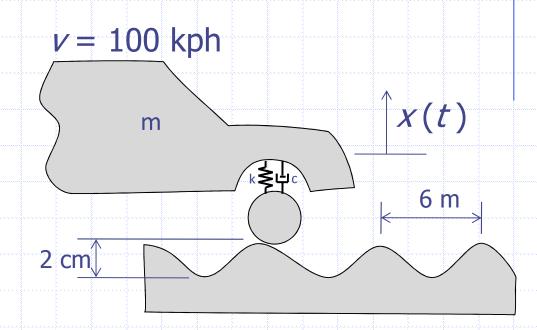


Base Excitation Section 2.4

(Skipping Section 2.3)

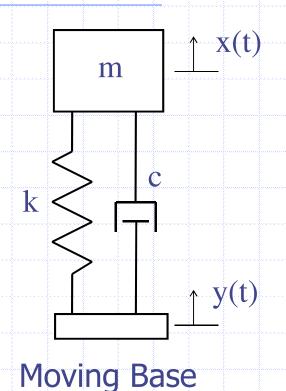
Vehicle traveling down bumpy road

- Given:
 - = m = 1007 kg
 - $k = 4 \times 10^5 \text{ N/m}$
 - $c = 20 \times 10^3 \text{ Ns/m}$



- Find:
 - Amplitude of steady-state vibration: X
 - Amplitude of transmitted force: F_T

Equations of Motion for Moving Base



Free body diagram



Equations of Motion for Moving Base

If
$$y(t) = Y \sin \omega t$$

and $\dot{y}(t) = \omega Y \cos \omega t$

then

$$m\ddot{x} + c\dot{x} + kx = cY\omega\cos\omega t + kY\sin\omega t$$

or

$$\ddot{x} + 2\varsigma\omega_n\dot{x} + \omega_n^2x = 2\varsigma\omega_n\omega Y\cos\omega t + \omega_n^2Y\sin\omega t$$



Solving the Differential Equation

Solving in the same way as for F₀cosωt yields:

$$x_{p}(t) = \frac{2\varsigma\omega_{n}\omega Y}{\sqrt{(\omega_{n}^{2} - \omega^{2})^{2} + (2\varsigma\omega_{n}\omega)^{2}}}\cos(\omega t - \theta_{1})$$

$$+ \frac{\omega_{n}^{2}Y}{\sqrt{(\omega_{n}^{2} - \omega^{2})^{2} + (2\varsigma\omega_{n}\omega)^{2}}}\sin(\omega t - \theta_{1})$$

$$+ \frac{(\omega_{n}^{2} - \omega^{2})^{2} + (2\varsigma\omega_{n}\omega)^{2}}{\sqrt{(\omega_{n}^{2} - \omega^{2})^{2} + (2\varsigma\omega_{n}\omega)^{2}}}\sin(\omega t - \theta_{1})$$

where:

$$\theta_1 = \tan^{-1} \left(\frac{2 \varsigma \omega_n \omega}{\omega_n^2 - \omega^2} \right)$$

Solving the Differential Equation

$$x_{p}(t) = \omega_{n} Y \left[\frac{\omega_{n}^{2} + (2\varsigma\omega)^{2}}{(\omega_{n}^{2} - \omega^{2})^{2} + (2\varsigma\omega_{n}\omega)^{2}} \right]^{\frac{1}{2}} \cos(\omega t - \theta_{1} - \theta_{2})$$

where:
$$\theta_2 = \tan^{-1} \left(\frac{\omega_n}{2\varsigma \omega} \right)$$

Solving the Differential Equation

• Final Equation (for base excitation without free vibration):

$$x_p(t) = X \cos(\omega t - \theta_1 - \theta_2)$$

where:

$$X = Y \left[\frac{1 + (2\varsigma r)^2}{(1 - r^2)^2 + (2\varsigma r)^2} \right]^{\frac{1}{2}}$$

$$r = \frac{\omega}{\omega_n}$$

$$\theta_1 = \tan^{-1} \left(\frac{2\varsigma r}{1 - r^2} \right)$$

$$\theta_2 = \tan^{-1} \left(\frac{1}{2\varsigma r} \right)$$

Plotting the "Displacement Transmissibility"

• Looking at just the steady-state vibration $x_p(t)$, we can plot the ratio of the amplitude X versus the amplitude Y as a function of base frequency ω .

How much force is transmitted?

$$F(t) = k(x - y) + c(\dot{x} - \dot{y}) = -m\ddot{x}(t)$$

$$-m\ddot{x}(t) = -m\frac{d^2x_p(t)}{dt^2} = m\omega^2Y \left[\frac{1 + (2\varsigma r)^2}{(1 - r^2) + (2\varsigma r)^2} \right]^{\frac{1}{2}} \cos(\omega t - \theta_1 - \theta_2)$$

Final equation (for transmitted force due to steady-state vibration):

$$F(t) = F_T \cos(\omega t - \theta_1 - \theta_2)$$

$$F_T = kYr^2 \left[\frac{1 + (2\varsigma r)^2}{(1 - r^2)^2 + (2\varsigma r)^2} \right]^{\frac{1}{2}}$$

Plotting the "Force Transmissibility"

• Looking at just the forced vibration $x_p(t)$, we can plot the ratio of the amplitude of the dynamic transmitted force F_T versus the static force kY as a function of base frequency ω .